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# The Differential Pressure numerical model



## Webinar

20 February 2023

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# The SAFEST consortium



The UNICAS task is related to the investigation of the interaction between the test liquid and the flowmeter's performance through numerical simulations.

Development and validation of a CFD numerical tool to support experimental campaigns, aiming at:

- *investigating different operative conditions (e.g. flow-rate, temperature, pressure);*
- *performing parametric analysis, which is difficult to carry out experimentally;*
- *obtaining detailed information about thermo-fluid dynamic fields (e.g. liquid densities, viscosities, etc.)*

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# Computational Fluid Dynamics



Computational Fluid Dynamics is a very effective tool for understanding fluid flow properties and predicting how the flow will respond to different boundary conditions.

The Navier-Stokes equations are turned in CFD application into systems of linear equations:

(1) Conservation of mass equation

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0$$

(2) Conservation of momentum equation

$$\frac{\partial(\rho \mathbf{u})}{\partial t} + \mathbf{u} \cdot \nabla(\rho \mathbf{u}) = \nabla \cdot \boldsymbol{\sigma} + \rho \mathbf{f}$$

(3) Conservation of energy equation

$$\frac{\partial E_t}{\partial t} + \nabla \cdot (E_t \mathbf{u}) = \nabla \cdot \boldsymbol{\sigma} \mathbf{u} - \nabla \cdot \mathbf{q}$$

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# CFD application within the project



The developed CFD numerical tool allows to :

- *predict fluid behaviour;*
- *calculate detailed pressure and velocity fields;*
- *investigate different boundary conditions and fluid properties.*

The study case is an orifice plate flow meter, whose analysis has been conducted through OpenFOAM which is a Finite-Volume-Method, open-source and completely programmable software.

Open  FOAM®



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# Numerical Model



The study case is a three-dimensional, incompressible and time-variant case.

A summary table is presented, containing the numerical simulation details.

	<b>Numerical model</b>
<b>Mathematical Model</b>	PISO (Pressure Implicit with Splitting Operators)
<b>Time Step</b>	Adjustable Time Step
<b>Courant Number</b>	2
<b>Residual Control</b>	$10^{-6}$
<b>Turbulence Model</b>	Realizable $k - \varepsilon$

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## Test setup

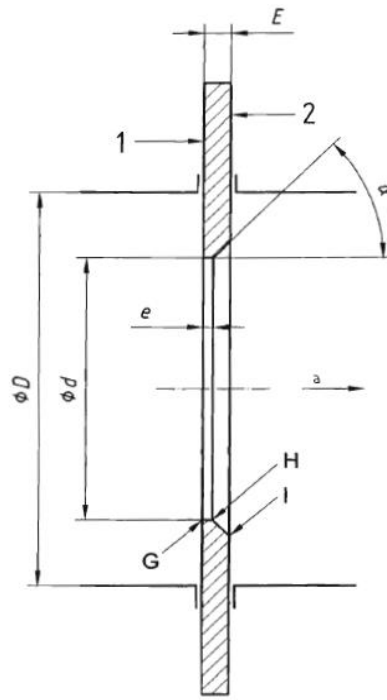


The case setup has been defined considering the following specifications and evaluating the response of the numerical model varying the inlet boundary conditions:

<b>Test setup</b>	
<b>Test liquids</b>	white spirit 180 EA (D 60)
<b>Pipe diameter</b>	DN 50
<b>Flow-rate range:</b>	3 m <sup>3</sup> /h -8 m <sup>3</sup> /h
<b>Pressure:</b>	ca. 3 bar
<b>Temperature (test liquid):</b>	25, 30 and 40 °C
<b>Temperature (ambient):</b>	room

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How it has  
been chosen?



The ISO Standard 5167-2 imposes the geometry of the orifice plate.

Parameters	Case study
Pipe Diameter $D$	$D = 5.476 \cdot 10^{-2} m$
Orifice Diameter $d$	$d = 1.847 \cdot 10^{-2} m$
Thickness $e$	$e = 5 \cdot 10^{-4} m$
Thickness of the plate, $E$	$E = 2 \cdot 10^{-3} m$
Angle of bevel $\alpha$	$\alpha = 45^\circ$

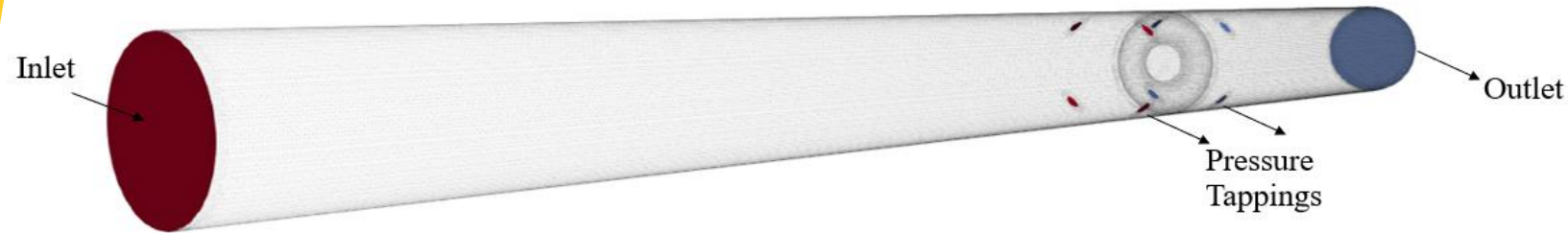


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CFD  
application  
within the  
project



The computational domain is presented in the figure below, together with the main boundary conditions:



Boundary conditions	Velocity	Pressure
Inlet	Flow-rate 1.084 [kg/s]	3 [bar]
Outlet	$\nabla u = 0$	0
Pressure Tappings	$\vec{U} = 0$	$\nabla p = 0$

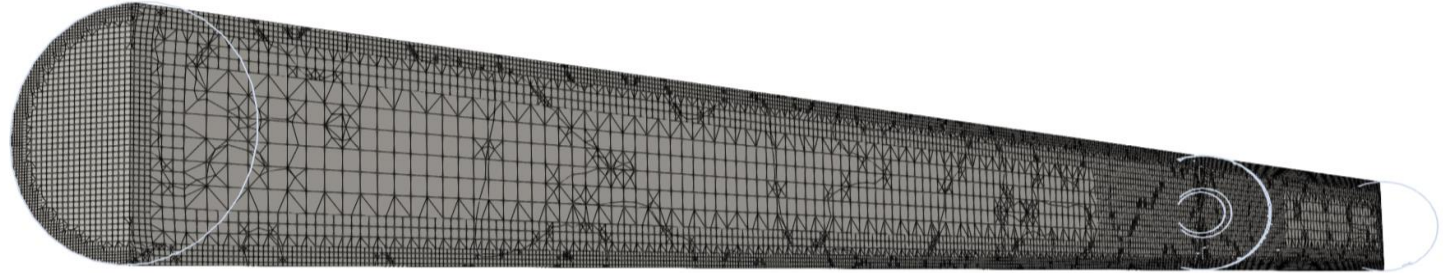


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Numerical  
domain



For the sensitivity analysis a 3.2 million cells reference grid has been adopted, comparing it with two mesh grids of 1.6 million and 800 thousand cells. The chosen grid is presented in the table below:



	<b>Characteristics</b>
<b>Number of Elements</b>	$1.6 \cdot 10^6$ cells
<b>Skewness</b>	2.387
<b>Non-orthogonality</b>	60.01

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# Fluid properties



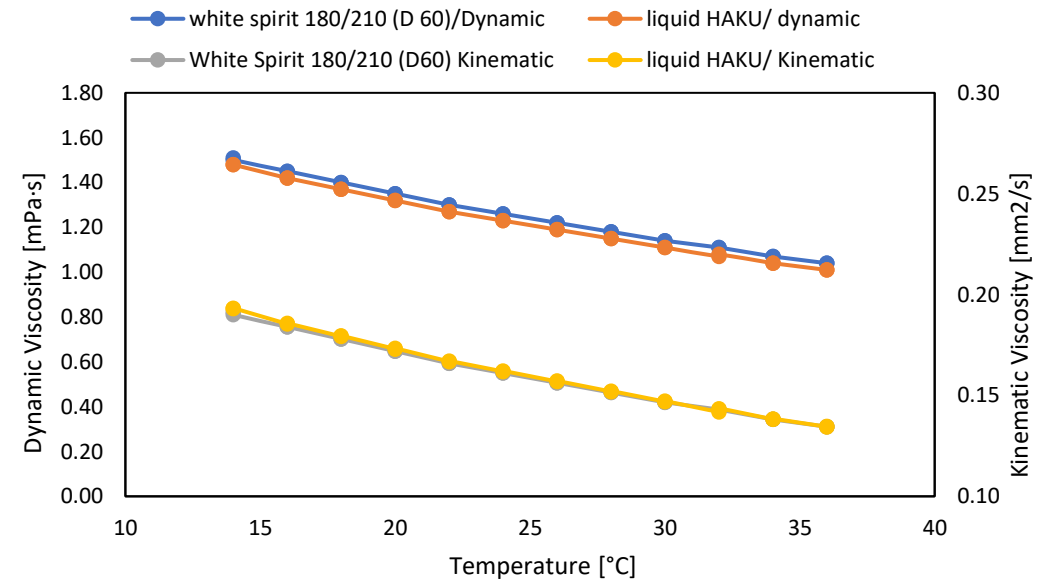
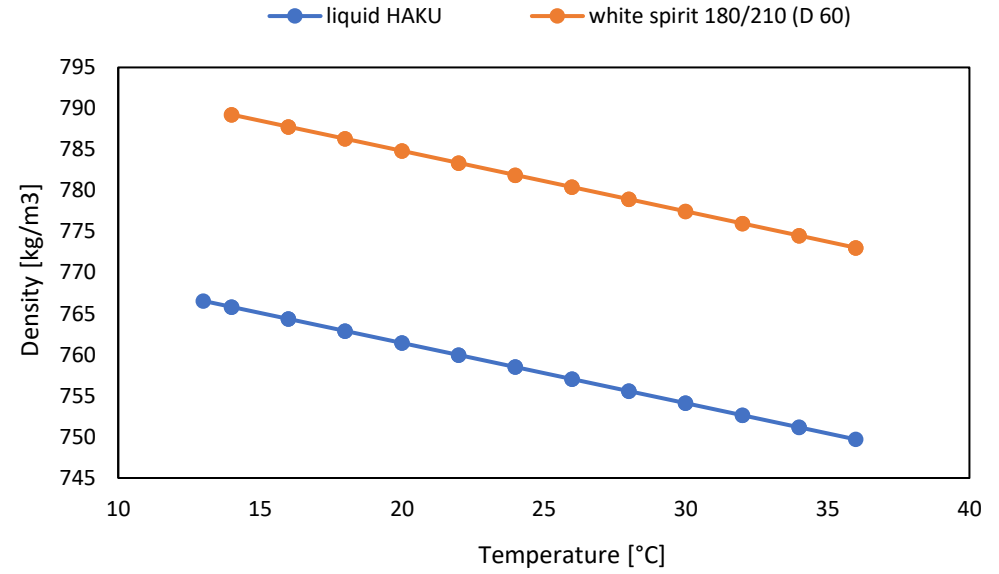
Different fluid operating conditions have been simulated, aiming at evaluating deviation from the reference conditions and at testing the numerical model.

The fluid properties at different temperatures are reported in the table:

<b>WHITE SPIRIT 180</b>			
<b>Fluid Temperature [°C]</b>	<b>25</b>	<b>30</b>	<b>40</b>
<b>Density [kg/m<sup>3</sup>]</b>	781.13	777.453	750.08
<b>Dynamic Viscosity [Pa*s]</b>	$1.24 \cdot 10^{-3}$	$1.14 \cdot 10^{-3}$	$1.02 \cdot 10^{-3}$
<b>Kinematic Viscosity [m<sup>2</sup>/s]</b>	$1.59 \cdot 10^{-6}$	$1.47 \cdot 10^{-6}$	$1.36 \cdot 10^{-6}$

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## Fluid properties



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# Turbulence Model



The three different turbulence models below have been considered to evaluate the performance of the numerical model.

	Realizable k-ε	k-ω SST	k-ε
$\Delta p_{\text{Transient}}$ [mbar]	223.41	222.25	219.82
$\Delta p_{\text{Theory}}$ * [mbar]		223.17	
Error %	0.19%	1.64%	2.71%

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## Results



The averaged field approach is considered to evaluate the performance of the orifice flow meter. A maximum fluctuation of 3% from the average value is considered to set the average time.

The results are expressed in terms of:

- *Pressure and velocity fields at different temperatures;*
- *Velocity profiles at 25°C;*
- *Comparison of pressure drops at different temperatures with the ISO 5167-2;*

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## Results



The results of the numerical simulations for three different temperature tests are presented, all other factors held constant. Two comparison are reported:

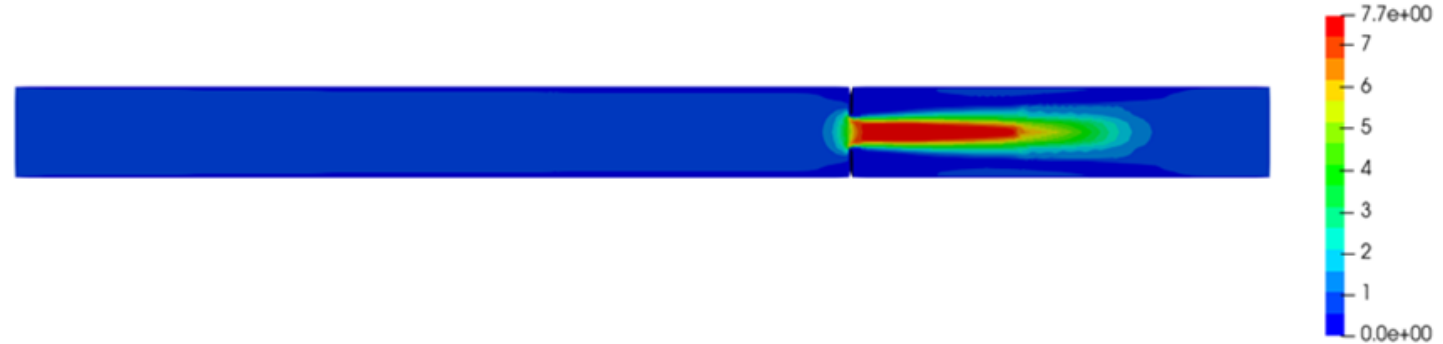
- *Velocity field comparison for 25°C, 30°C and 40°C;*
- *Pressure Field Comparison for 25°C, 30°C and 40°C.*

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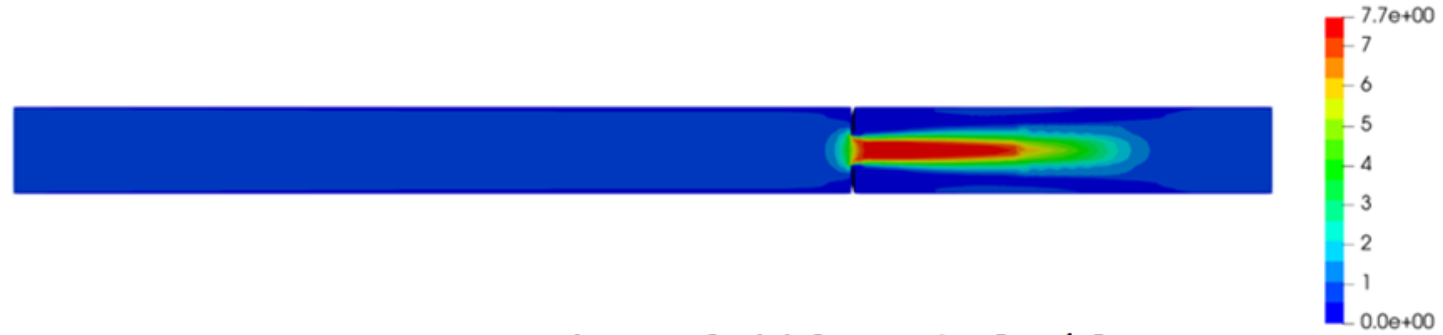
Velocity field



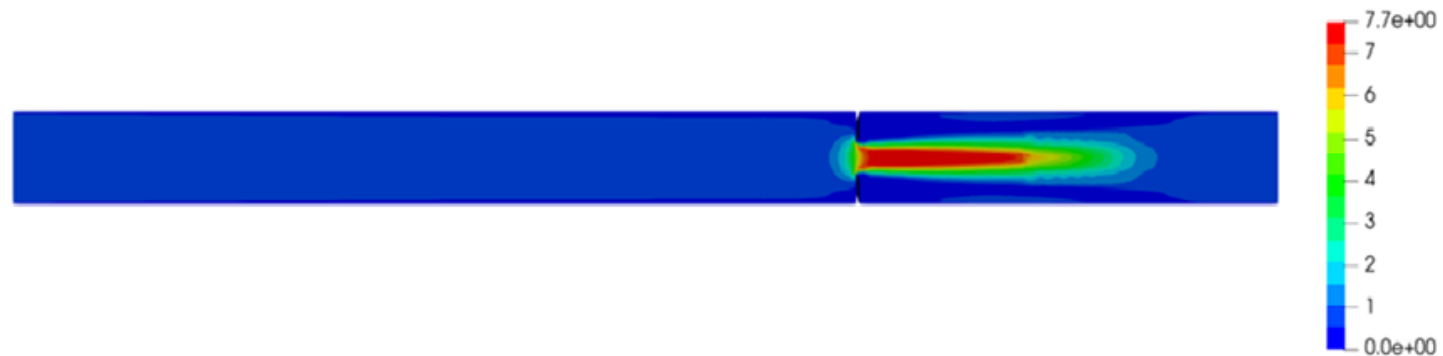
Average velocity field for 25°C [m/s]



Average velocity field for 30°C [m/s]



Average velocity field for 40°C [m/s]

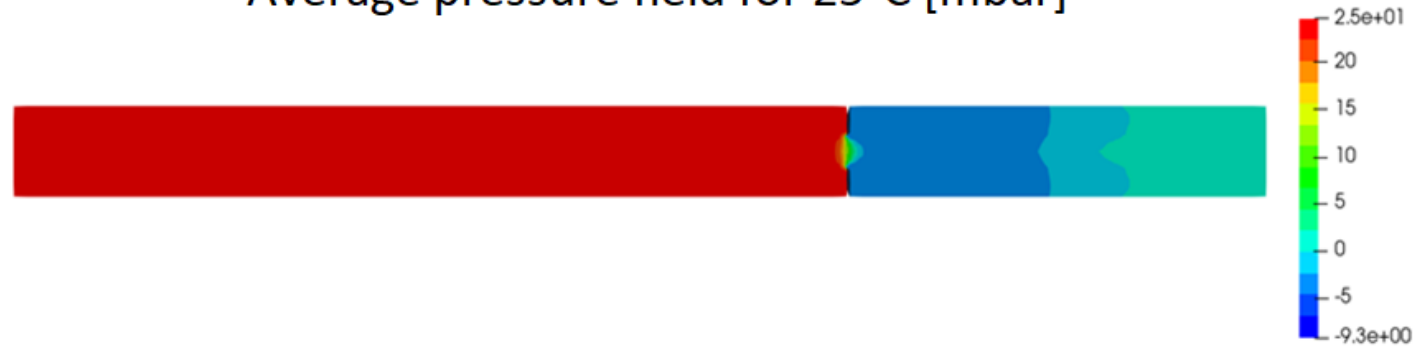


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Pressure field



Average pressure field for 25°C [mbar]



Average pressure field for 30°C [mbar]



Average pressure field for 40°C [mbar]





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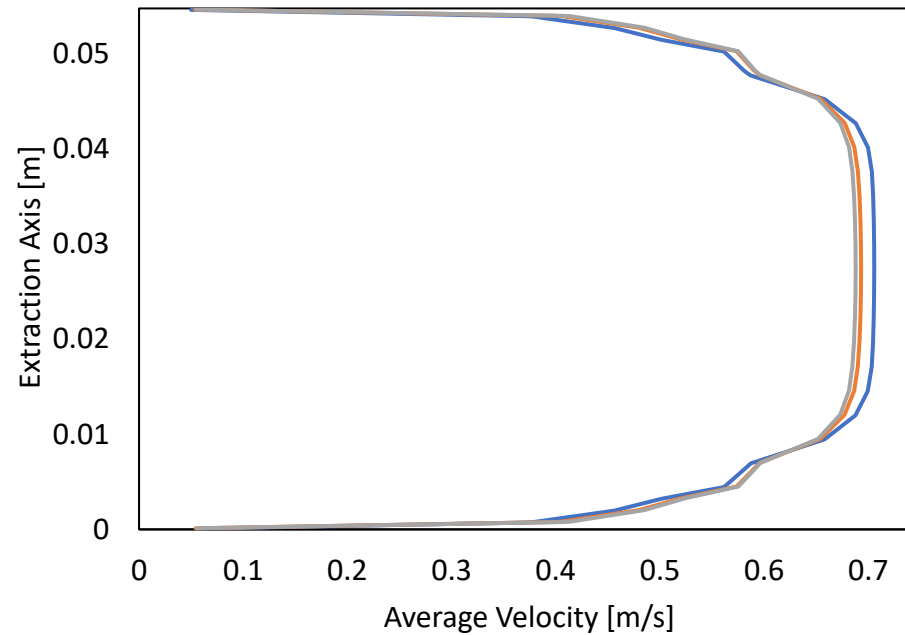
# Velocity profiles



The velocity profiles have been extracted in correspondence of pressure tappings. The pressure taps section are located:

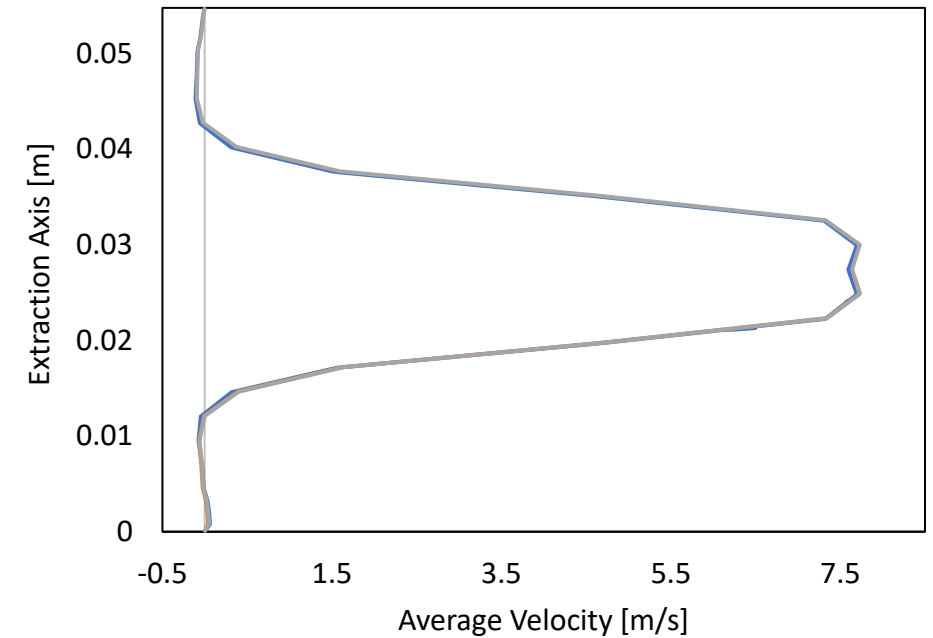
- One diameter before the orifice plate;
- Half diameter after the orifice plate;

Average Velocity Profile - Before Orifice plate



— T-25°C — T-30°C — T-40°C

Average Velocity Profile - After Orifice plate



— T-25°C — T-30°C — T-40°C

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Numerical  
tool  
validation



The pressure drop calculated through the numerical analysis has been compared with the theoretical pressure drop of the ISO standard 5167-2.

	Test 25°C	Test 30°C	Test 40°C
$\Delta p_{\text{Numerical Analysis}}$ [mbar]	223.41	225.73	217.94
$\Delta p_{\text{Theory}}^*$ [mbar]	223.17	222.33	214.50
Error %	0.19%	1.53%	1.60%

\*Theoretical Differential pressure ISO 5167-2

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## Conclusions and future developments



A numerical model of the orifice flow meter has been developed, following the ISO standard 5167-2.

The mesh and turbulence sensitivity analysis have been conducted to provide the best model possible and reduce the average error from the reference.

A numerical validation has been conducted against the pressure drop obtained through the iterative calculation of ISO standard 5167-2.

Future Developments of this research:

- *Experimental validation of the numerical model, through an experimental campaign;*
- *Variable flow-rate profile to inlet section;*